AMENDMENTS TO THE SPECIFICATION

IN THE TITLE OF THE INVENTION:

Please amend the title as it appears on the first page of the specification and in the U.S. Patent and Trademark Office's records, as follows:

--~~MAGNETIC BEARING ASSEMBLY~~HEAT-DISSIPATING DEVICE--

IN THE SPECIFICATION:

Please amend the paragraph beginning on page 1, line 14, as follows:

--Please refer to Figs. 1(a) and 1(b). The ball bearing 1 includes an inner ring 13 which includes a groove as a raceway, an outer ring 12 which also includes a groove as a raceway and a plurality of balls 11 which are inserted between respective grooves of the inner and outer rings. The inner ring 13 is usually fixed to a rotary shaft 21 of a rotor, and the outer ring 12 is fixed to a base or a stator 23. The upper side of the ball bearing 1 is urged against a spring 22 for facilitating smooth rotation.--

Please amend the paragraph beginning on page 1, line 22, as follows:

--Upon rotating the shaft 21, the balls 11 of the ball bearing 1 race around the grooves inward the outer ring 12 and outward the

inner ring 13. After the ball bearing 1 is used for a certain period of time, the fatigue of the metal material is customarily found, thereby causing the balls 11 and the surface of the inner ring 13 or the outer ring 12 to be abraded.--

Please amend the paragraph beginning on page 1, line 27, as follows:

--Referring to Figs. 2(a) and 2(b), a self-lubricating bearing 3 manufactured of polymeric material is usually in a shape of a sleeve ~~3~~ and it contains minute passages or channels carrying therein the lubricating oil 31 such that the oil 31 can be deposited on the shaft 21 by diffusion into the inner wall 32 upon rotation of the shaft 21. The inner wall 32 of the self-lubricating bearing 3 usually contacts with a rotary shaft 21, and the outer wall 33 is fixed to a base 23.--

Please amend the paragraph beginning on page 2, line 7, as follows:

--In contrast, the ball bearing 1 can perform under heavy loads and has a long life; however, it is costly and has the disadvantage of being abraded. The self-lubricating bearing 3 has good self-lubricating properties to reduce abrasion and is cheaper than the ball bearing; however, it generally incapable of being operated with large loads and its life is not very long.--

Please amend the paragraph beginning on page 4, line 15, as follows:

--Figs. 3(a) to 3(c) ~~is a~~ are schematic diagrams of the magnetic bearing assembly applied to a cooling fan according to the first embodiment of the present invention. The cooling fan includes a rotor 6, and a base 23. The rotor 6 has an impeller 61 and a shaft 21. The base 23 is for supporting the rotor 6. The magnetic bearing assembly includes a magnetic portion and a bearing portion. The bearing portion is a sleeve bearing 5. The magnetic portion includes an upper magnetic portion constituted by three magnetic rings 51, 52 and 53, and a lower magnetic portion composed of three magnetic rings 511, 521 and 531. In the upper magnetic portion, the first magnetic ring 51 and the third magnetic ring 53 are connected to the base 23 of a frame 4 of the cooling fan and the second magnetic ring 52 is connected to the shaft 21. The second magnetic ring 52 and the third magnetic ring 53 are disposed in a radial alignment, wherein these two rings are assembled with each other to have the same polar disposition for generating the repulsive magnetic field. In addition, the first magnetic ring 51 and the second magnetic ring 52 are disposed in an axial alignment to have an opposite polar disposition for generating an axially repulsive magnetic field. The radially repulsive magnetic field generated between the second magnetic ring 52 and the third

magnetic ring 53, and the axially repulsive magnetic field generated between the first magnetic ring 51 and the second magnetic ring 52 allow to reduce the friction between the sleeve bearing 5 and the shaft 21 upon rotation of the shaft (as shown in Fig. 3(a)). Alternatively, the three magnetic ring 51, 52, and 53 can have different disposition, wherein said axially magnetic forces are repulsive magnetic forces and radially magnetic forces are attractive magnetic forces (as shown in Fig. 3(b)). Likewise, the three magnetic rings 511, 521 and 531 in the lower magnetic portion can effectively reduce the friction between the sleeve bearing 5 and the shaft 21 upon the operation of the cooling fan.--

Please add a paragraph before the paragraph beginning on page 5, line 11, as follows:

--Otherwise, the first magnetic ring 51, 511 and the second magnetic ring 52, 521 are disposed in an axial alignment with each other to have an opposite polar disposition (as shown in Fig. 3(c)).--

Please amend the paragraph beginning on page 5, line 17, as follows:

--Figs. 4(a) to 4(c) ~~is a~~ are schematic diagrams of the magnetic bearing assembly applied to the cooling fan according to

the second embodiment of the present invention. The cooling fan includes a rotor 6, and a base 23. The rotor 6 has an impeller 61 and a shaft 21. The base 23 is for supporting the rotor 6. The magnetic bearing assembly includes a magnetic portion and a bearing portion. The bearing portion is a sleeve bearing 5. The magnetic portion includes an upper magnetic portion having an inner magnetic ring 73 and an outer magnetic ring 74, and a lower magnetic portion having three magnetic rings 75, 76 and 77. In the upper magnetic portion, the inner magnetic ring 73 is connected to the shaft 21 and the outer magnetic ring 74 is connected to the base 23 of the frame of the cooling fan. These two magnetic rings 73 and 74 are disposed in a radial alignment with each other to have the same polar disposition for generating a repulsive magnetic force. In the lower magnetic portion, the first magnetic ring 75 and the third magnetic ring 77 are connected to the shaft 21 and the second magnetic ring is connected to the base 23. These three magnetic rings 75, 76 and 77 are disposed in an axial alignment to have the opposite polar disposition for generating the axially repulsive magnetic forces (as shown in Fig. 4(a)). Alternatively, the inner magnetic ring 73 and the outer magnetic ring 74 can be disposed in a radial alignment with each other to have an opposite polar disposition (as shown in Fig. 4(b)). Likewise, the first magnetic ring 75, the second magnetic ring 76 and the third magnetic ring 77 can also be disposed in an axial alignment with each other to have

an identical polar disposition for generating axially attractive magnetic forces (as shown in Fig. 4(c)). Therefore, the friction between the sleeve bearing 5 and the shaft 21 upon the rotation of the cooling fan is considerably reduced.--